

Analysis and Study of Heat Transfer Resistance Inside and Outside the Reentry Capsule

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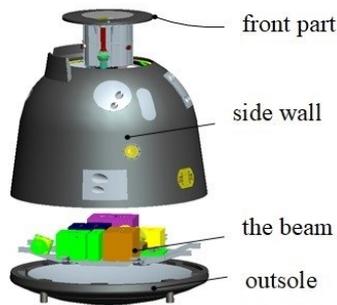
Abstract: According to the heat transfer characteristics inside and outside of the lunar-earth high-speed reentry capsule, a typical calculation model of heat conduction in external thermal protection system(TPS) coupled with internal radiation was established. The thermal properties of thermal resistance inside and outside the reentry capsule were analysed. The effects of thickness of the TPS, surface conditions and atmospheric pressures on the temperature were further explored. The results showed that atmosphere pressure was necessary to be controlled under 10Pa to ensure the safety temperature of the equipment and pipe. Based on the critical pressure, the configuration was optimized. The results provide detailed data for the system design of the lunar exploration, and also provide a reference for the thermal design of the atmospheric reentry spacecraft.

1 Introduction

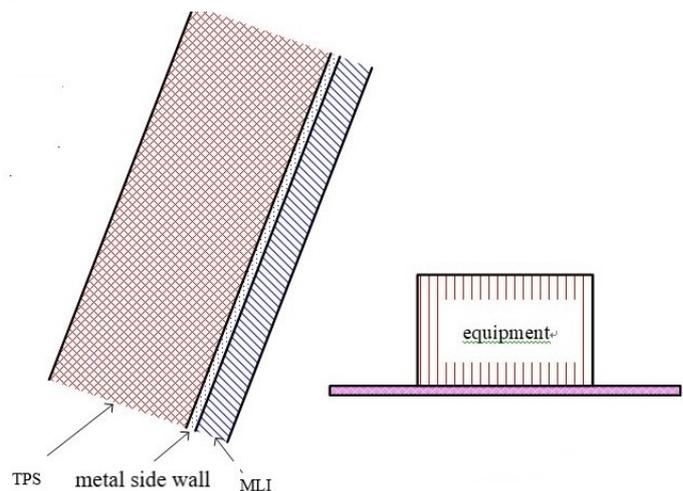
In order to prevent the pneumatic heat eroding the inside wall in the return process, the outer surface of lunar-earth high-speed reentry capsule (reentry capsule) was fully coated with thermal protection system(TPS)¹. So the enclosure formed a closed chamber, and the internal and external gas environment was unsmooth, and stated in a medium vacuum environment for long time.

The maximum heat insulation design² is used inside and outside the reentry capsule. Figure 1(a) shows the overall shape of the reentry capsule. The outer mounted TPS is different from the recoverable satellite³ manned

return module⁴, the reentry capsule is thicker and the thickness range is 28~66.5mm. The thermal resistance of the thermal protection has a great influence on the thermal resistance of the whole re-entry capsule. The metal side wall and beam are covered by multilayer heat insulation (MLI) inside to ensure the small heat leakage in the in-orbit storage stage and to restrain the high enthalpy during the return process. The external and internal thermal resistance affect each other. Therefore, it is important to analyze the influence of thermal resistance of each layer on the temperature level of the whole reentry capsule.



(a).overall shape of reentry capsule



(b).physical model

Figure 1. Configuration of reentry capsule.

2 Reentry Capsule Heat Transfer Model

2.1 Physical Model

Heat transfer resistance inside and outside the reentry capsule includes thermal resistance of TPS, thermal resistance of metal structure, multilayer radiation resistance and beam in medium vacuum environment, thermal resistance of equipment, etc. Figure 1(b) shows the installation of the side wall and the equipment. The TPS is installed on the aluminum plate. Inside of the aluminum plate is covered with MLI, the surface of the equipment is blackened, and the MLI and equipment are radiative heat transfer. The pipe diameter Φ 6mm, pipe clamps are made of non-metallic polyimide material and pipe mounting seat is aluminum base, welding with side wall.

2.2 Mathematical Model

The physical heat transfer process can be simplified as follows:

- (1) The convection heat transfer can be neglected, due to the small space in the device.
- (2) The influence of cable and cable support on equipment occlusion is not considered.
- (3) The thickness of the metal side wall is only 0.5 mm. It is considered that the temperatures of inner and outer layer of the MLI are equal.

Based on the above analysis, mathematical model is established as follows,

- 1) Temperature outside the thermal insulation structure

$$m_f C_{pf} \frac{dT_f}{d\tau} = Q_f + K_{af} A_h (T_a - T_f) - \varepsilon_f \sigma A_h T_f^4 \quad (1)$$

- 2) Multilayer temperature

$$m_a C_{pa} \frac{dT_a}{d\tau} = K_{ah} A_h (T_h - T_a) - K_{af} A_h (T_a - T_f) \quad (2)$$

- 3) Multi-layer inner temperature

$$m_h C_{ph} \frac{dT_h}{d\tau} = K_{ah} A_h (T_h - T_a) - \frac{\sigma A_{nr}}{1/\varepsilon_n + A_{nr}(1/\varepsilon_h - 1)A_h} (T_n^4 - T_h^4) \quad (3)$$

(3)

- 4) Equipment temperature

$$m_n C_{pn} \frac{dT_n}{d\tau} = Q_n - Q_w - \frac{\sigma A_{nr}}{1/\varepsilon_n + A_{nr}(1/\varepsilon_h - 1)A_h} (T_n^4 - T_h^4) \quad (4)$$

m_n is quality of equipment, m_a is MLI bulkhead side mass, m_h is MLI inside side mass, m_f is quality of the TPS, Q_f is the absorbed external heat flux, C_{pn} is specific heat capacity of equipment, C_{ph} is MLI chamber film heat capacity, C_{pa} is MLI bulkhead side heat capacity, C_{pf} is heat capacity of TPS, A_{nw} is contact area between equipment and mounting plate, A_h is MLI area, A_f is the TPS area, ε_n is surface emissivity of equipment, ε_h is MLI emissivity, ε_f is TPS emissivity, K_{ah} is MLI heat transfer coefficient, K_{af} is internal and external heat transfer coefficient of the TPS.

3 Analysis of the Thermal Characteristics of the Reentry Capsule

3.1 Analytical Model Parameters and Boundary Conditions

According to the characteristics of the overall configuration of the reentry capsule, the structural design and the system parameters, the analytical model parameters are determined, as shown in table 1. In the analysis, the cruising section with the worst low temperature of the reentry capsule is selected for analysis. The material parameters are taken at the beginning of the life, and the external heat flux is applied to the ship. The laser IMU and the typical pipe is selected for analysis and research.

Table 1. Analysis of model parameter values.

Code	Values	Code	Values	Code	Values
m_n	7.65kg (laser)	A_{nr}	0.185m ²	Q_f	128W
$m_{a(h)}$	50.4g	A_{nw}	0.042m ²	Q_n	16W
m_f	25.2Kg	A_h	5.04m ²	C_{pn}	835J/Kg.K
m_w	0.93kg	A_f	5.04m ²	C_{ph}	600J/Kg.K
K_{af}	0.00524W/m ² .K	A_w	0.15m ²	C_{pa}	600J/Kg.K
ε_w	0.1	ε_n	0.88	C_{pf}	1.4×10 ⁻³ J/kg.K
ε_f	0.87	ε_h	0.1	C_{pw}	900J/Kg.K

3.2 Results and Analysis of Influencing Factors

All painted white paint on the outside of the reentry capsule, the required heat of the propulsion system pipe is 16W2. In cruise attitude flight, only the top surface is radiated by external heat flux, and the absorption of external heat flux is relatively small. During cruise flight, the two IMU should meet the requirements of starting temperature 0°C, and the pipe temperature (T_{pipe}) should be maintained above 5°C. The steady-state results are as follows: the temperature of IMU (T_{IMU}) is 2.9 °C, and the temperature of typical pipe is 5.9 °C.

In the thermal resistance inside and outside the device, the MLI thermal resistance and the thermal resistance of the TPS are the two most important thermal

resistances. The decrease of true test of MLI environment has a great influence on MLI performance⁵⁶. The thermal resistance of TPS mainly depends on the thickness of the TPS. Therefore, the influence of MLI vacuum degree and thermal insulation thickness on the temperature of electronic equipment and pipe is evaluated by setting up the analysis condition of the influence of the vacuum degree and the thickness of the TPS, respectively.

Influence of thickness of the TPS

Under other conditions, the thickness of the TPS is reduced to 3~16mm from the original 33mm, and the results are shown in table 2. It can be seen that when the thickness of the TPS is reduced to 11mm, the temperature of the laser IMU will be less than 0°C.

Table 2. Thickness of TPS.

Working condition	Thickness/mm	$T_{IMU} / ^\circ C$	$T_{pipe} / ^\circ C$
Case 1	0.033	2.9	5.9
Case 2	0.016	0.7	5.8
Case 3	0.011	-0.1	5.6
Case 4	0.0055	-0.4	5.2
Case 5	0.0003	-1.2	4.6

(1) Vacuum effect

T_{IMU} and T_{pipe} obtained by steady-state calculation under different vacuum conditions are shown in table 3. With the pressure increasing, T_{IMU} and T_{pipe} is reduced. When the pressure is greater than 10Pa, T_{pipe} is lower than 5°C.

When the pressure is more than 50Pa, T_{IMU} is lower than 0°C. The temperature sensitivity of propulsion pipeline to vacuum degree is stronger than that of equipment because of the influence of multiple layers of bulk coating and multi-layer of side wall.

Table 3. Different vacuum analysis conditions.

Working condition	Internal pressure/Pa	$T_{IMU} / ^\circ C$	$T_{pipe} / ^\circ C$
Case 6	0.001	15.3	9.4
Case 7	0.01	9.1	7
Case 8	1	1	6.4
Case 9	10	2.9	5.9
Case 10	50	0.6	4.4
Case 11	100	-2.6	3.2

(2) Effect of MLI surface condition

The influence of infrared emissivity is considered in the MLI of the device, and the polyester film, polyimide film and F46 face film are set up respectively.

It can be seen from table 4 that the heat transfer between the equipment and the bulkhead increases and the

equipment temperature decreases with the increase of the MLI emissivity in the device. The reentry device is in the cruise section and the reentry section is designed for heat insulation. During the two flights, the heat control power is less. Therefore, the selection of polyester film on the multi-layer surface is the most favorable.

Table 4. MLI surface condition.

Working condition	MLI surface shape	$T_{IMU} / ^\circ C$
Case 12	Polyester film, emissivity 0.12	2.9
Case 13	F46 surface, emissivity 0.68	-13.9
Case 14	Polyimide film, emissivity 0.88	-15.8

4 Structural Optimization

According to the results in Table 3, if T_{pipe} can be controlled above $5^\circ C$, the internal pressure value of the device cannot be greater than 10Pa. Since the pressure inside and outside of the return device is not smooth, it is necessary to design a suitable pressure relief hole without affecting the pneumatic, ablation and measurement and control, and to combine the idea of unified design of heat prevention and heat control grooming⁷⁸. Therefore, two Φ 6mm holes are designed at the front. The designed structure verifies the pressure relief ability of the through hole in the vacuum tank. After 7 hours of vacuum pumping, the pressure in the device decreases to 7.1 Pa.

In the heat balance test of the reentry capsule, two vacuum meters were installed at the inner side wall of the injector, and the vacuum degree in the device was monitored. The pressure reduced to 8.9Pa in 6 hours. During the thermal test, the vacuum degree inside the device was higher than 0.12Pa.

5 Conclusion

Based on equilibrium thermal mathematical model, the matching between thermal resistances was analyzed. The pressure relief of different vacuum degree was investigated by numerical simulation. The influence of different thickness of the TPS and different surface condition on the temperature level in the device can be concluded as follows. (1) The effect of vacuum degree on the MLI of the reentry capsule was more serious. If the

vacuum degree was greater than 10Pa, T_{pipe} would not meet the requirements of the index. (2) The thermal resistance of the TPS was large, when the average thickness of TPS was reduced from 33mm to 11mm, T_{IMU} would be lower than $0^\circ C$, which can not meet the starting temperature index. (3) The MLI surface condition has a great influence on the temperature of the device. If the F46 or polyimide film was chosen, the temperature of the device would drop sharply, which can not meet the temperature requirement. Therefore, polyester film is beneficial to the insulation of cruise section and reentry section.

6 Reference

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