

Motion Error Analysis and Modeling Technology of CNC Camshaft Grinder

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Abstract. Taking certain CNC camshaft grinder as the subject, its motion error analysis and modeling were studied. The movement forms and types of errors between the moving parts were analysed, the movement between coordinate systems of the adjacent bodies was used to express the movement between the two adjacent bodies, and the ideal motion equation and actual motion equation in case of errors between the adjacent bodies were set up. Actual motion equation between adjacent bodies was made and further extended to analyse arbitrary low-order body arrays, which provides a theoretical basis for studying the multi-branch CNC camshaft grinding machine error modeling. Complex multi-branch chain CNC camshaft grinder simplified was made as a simple multibody system, and corresponding body coordinate systems and motion reference coordinate systems for moving parts were established to calculate the corresponding transformation matrix between adjacent bodies. Moving parts of the machine were divided into “workpieces-bed” and “wheel-bed” two kinematic chains. The precision constraint equation of the machine is $P_t = P_w$ in case of using errors influence, and the constraint equation was solved. It provides necessary conditions for study of CNC camshaft grinder error compensation. Results show that the machine precision is significantly improved after error compensation.

1 Introduction

High machining precision is critically important and has increased demand in recent years in aerospace, military, ships, cars, and other industries [1,9]. CNC camshaft grinder as an efficiency, high-precision machining equipment of the camshaft which precision retaining ability is essential. However, Due to the existence of different kinds of error source, all machine tools will be faced with the problem of lower accuracy. It is necessary to analyze the errors of machine tools and take appropriate measures to realize error compensation. Therefore, the method of reducing the effect of error on the machining precision of machine tool can be divided into error prevention and error compensation. By establishing the error model of machine tools, identifying all errors of machine tools and reducing the influence of machine errors on machining accuracy by means of correcting NC instructions, which is called error compensation to reduce the effect of the error of machine tool on the machining precision of the workpiece. The method can overcome the shortcomings of the traditional error prevention method, such as high cost and poor applicability, so it has been developed rapidly.

The kinematic error analysis and modeling of machine tools are the basis of error compensation, the analysis of kinematic error is the key of error modeling. In order to improve the machining accuracy and precision retaining ability of the CNC camshaft grinder, the kinematic error analysis and modeling of the CNC camshaft grinder was

proposed. At present, considerable research work for error compensation of multi-axis machine tools is devoted. Su[1] adopted the multi body system (MBS) theory for the precision modeling and error compensation of multi axis CNC machine tools. Fan et al.[2] studied the geometric error compensation technique for improving the accuracy of precision cam grinding. According to ref.[3], the software design of the error analysis and compensation technology for the camshaft NC grinding are carried out. Wang[4]adopted polynomial fitting and linear fitting method to the geometric error and thermal error modeling of CNC machine tools and the on-line error compensation. According to ref.[5], the geometric error compensation and error traceability of CNC machine tools were analyzed. Lechniak et al.[6] proposed the offline software error compensation method. Chnan et al.[7] presented a error compensation method for geometric error and force error of three-axis grinding machine. Zhang et al.[8] used the double ball bar to detect and compensate the error of the rotary table of five axis CNC machine tool. Machine tools are regarded as complex mechanical systems which made up of multiple moving bodies[10]. In recent years, along with the application of follow-up type camshaft grinder is more extensive, in addition, the error analysis and modeling method is too complex and lack of general model. Therefore, the theory of multi-body system can be used to reduce the difficulty of the error modeling of machine tool. However, the multi body system theory is seldom applied to the error modeling of machine tool. So this

paper uses the theory of multi-body system to establish model of follow-up type camshaft grinder geometry error.

2. Kinematic error analysis of CNC camshaft grinder

In this study, a CNC camshaft grinder is chosen as the research object, as shown in Fig. 1. The machine tool consists of machine bed (body 1), Z-axis slide carriage (body 2), grinding wheel heads (body 3), spindle (body 4), workpiece (body 5), X-axis slide carriage (body 6), grinding wheel frame (body 7), grinding wheel (body 8) and tailstock (body 9). The grinding wheel moves along the X-axis, worktable moves along the Z-axis, and the spindle rotation around the C-axis. The cam profile is accomplished by simultaneously controlling the X-axis and the C-axis. Geometric errors are usually composed of position-independent geometric errors (PIGEs), and position-dependent geometric errors (PDGEs). PIGE's don't vary with the moving position of the machine tool, but PDGEs are the opposite ones. Fig.2 is the schematic diagram of PIGE's between linear axes. Based on the theory of the rigid body motion, each moving part of a machine tool has six DOFs in the Cartesian coordinate system. Hence, each moving axis of the CNC camshaft grinder has six PDGEs. Fig.3 shows six PDGEs of X-axis, where $\delta_i(x)$ and $\varepsilon_i(x)$ ($i = x, y, z$) represent the linear errors and the angular errors of X-axis along the i direction respectively. Therefore, all the geometric errors of the CNC camshaft grinder are listed, as shown in the Table 1.

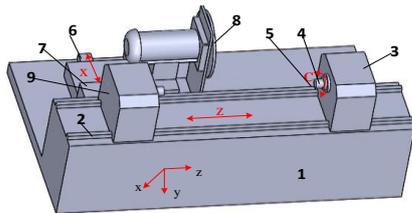


Fig. 1 Structure map of CNC camshaft grinder

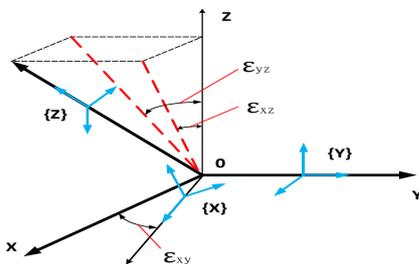


Fig.2 the schematic diagram of PIGE's between linear axes

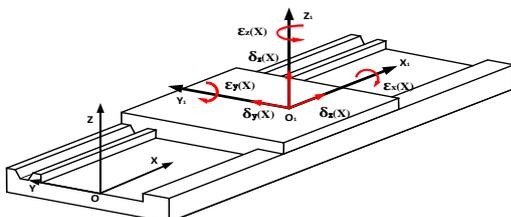


Fig.3 The schematic diagram of PDGEs of X-axis

3. Topological structure and lower body array

Based on MBS theory, a machine tool can be assumed to be composed of various rigid bodies. Fig.4 shows the topology structure map of the CNC camshaft grinder, where " B_i " means the i -th body. As shown in Fig. 4, the kinematic chain of the CNC camshaft grinder can be divided into two branches. One is workpiece branch ($B_1 - B_2 - B_3 - B_4 - B_5$), and the other is wheel branch ($B_1 - B_6 - B_7 - B_8$).

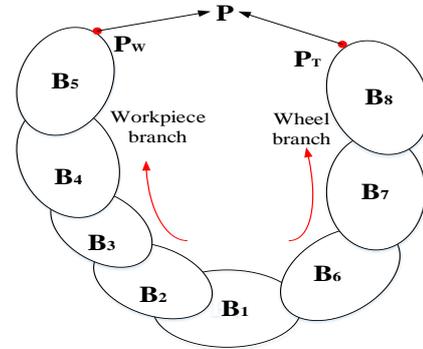


Fig.4 the topology structure of the CNC camshaft grinder

4. Establishment of the relationship between adjacent body motion

Suppose that the workpiece and tool are clamped without errors. Point O is origin of B_1 body coordinate and also base origin of the machine. When $j = 1, 2, 3, 4, 5, 6, Q_j^1$ and Q_j are ideal and actual motion reference points of B_j , respectively. O_j^1 and O_j are ideal and actual position origin of B_j body coordinate in Q_j coordinate system. Thus, q_j^1 and q_j are ideal and actual position vectors of B_j relative to B_{j-1} and q_j^e is position error vector. S_j^1 and S_j are ideal and actual motion vectors of B_j relative to B_{j-1} and S_j^e is displacement error. q_w is position matrix of workpiece relative to the worktable and q_t is position matrix of tool relative to B-axis. r_w and r_t are position vectors of tool center in the workpiece coordinate system and tool coordinate system.

Table. 1 Geometric errors of the CNC camshaft grinder

	Axis	Linear error			Angular error		
		x direction	y direction	z direction	Around x	Around y	Around z
PDGES	X	$\delta_x(x)$	$\delta_y(x)$	$\delta_z(x)$	$\varepsilon_x(x)$	$\varepsilon_y(x)$	$\varepsilon_z(x)$
	Z	$\delta_x(z)$	$\delta_y(z)$	$\delta_z(z)$	$\varepsilon_x(z)$	$\varepsilon_y(z)$	$\varepsilon_z(z)$
	C	$\delta_x(c)$	$\delta_y(c)$	$\delta_z(z)$	$\varepsilon_x(c)$	$\varepsilon_y(c)$	$\varepsilon_z(c)$
PIGES						$\varepsilon_{xz}, \varepsilon_{xc}$	ε_{zc}

5. Kinematic error modeling of CNC camshaft grinder

The CNC camshaft grinder is a special multi body system with only two branches, and the components are

$$P_w = P_{wo} = S_{21}S_{32}S_{43}S_{54}S_{5r} = S_{12p}S_{12pe}S_{12s}S_{12se}S_{23p}S_{23pe}S_{23s}S_{23se}S_{34p}S_{34pe}S_{34s}S_{34se}S_{45p}S_{45pe}S_{45s}S_{45se}S_{5r} \quad (3)$$

$$P_t = P_{to} = S_{61}S_{76}S_{87}S_{8r} = S_{16p}S_{16pe}S_{16s}S_{16se}S_{67p}S_{67pe}S_{67s}S_{67se}S_{78p}S_{78pe}S_{78s}S_{78se}S_{8r} \quad (4)$$

where P_w is actual grinding position in workpiece branch and P_t is actual Grinding position in tool branch. S_{ijp} is ideal position matrices, S_{ijs} is ideal motion matrices. S_{ijpe} is actual position matrices, S_{ijse} is actual motion matrices. S_{5r} represent position matrix of actual grinding point in workpiece branch, S_{8r} represent position matrix of actual grinding point in tool branch. An important ideal condition is that the actual grinding position in workpiece branch is equal to the actual grinding position in tool branch, as shown in Eq. 5,

$$P_w = P_t \quad (5)$$

connected in a certain form. Based on the theory of multibody system, the error modeling of the machine tool is established. the actual and theoretical tool center in the machine coordinate system can be obtained,

Suppose that the position matrix of the coordinate system of the workpiece relative to the coordinate system of the spindle faces is $(q_{5x} \ q_{5y} \ q_{5z} \ 1)^T$, the position matrix of the coordinate system of the tool relative to the coordinate system of the spindle faces is $(q_{8x} \ q_{8y} \ q_{8z} \ 1)^T$. Let E represents a unit matrix. Then, D-H matrices can be built, which are shown in Table 2.

Table 2. The D-H matrices of the CNC Camshaft Grinder (parts)

Ideal position and motion matrices	Actual position and motion matrices
$S_{12p} = E$	$S_{12pe} = E$
$S_{12s} = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$	$S_{12se} = \begin{bmatrix} 1 & -\varepsilon_z(z) & \varepsilon_y(z) & \delta_x(z) \\ \varepsilon_z(z) & 1 & -\varepsilon_x(z) & \delta_y(z) \\ -\varepsilon_y(z) & \varepsilon_x(z) & 1 & \delta_z(z) \\ 0 & 0 & 0 & 1 \end{bmatrix}$
$S_{23p} = E \quad S_{23s} = E$	$S_{23pe} = E \quad S_{23se} = E$
$S_{16p} = E \quad S_{16s} = E$	$S_{16pe} = E \quad S_{16se} = E$
$S_{67p} = E$	$S_{67pe} = \begin{bmatrix} 1 & 0 & \varepsilon_{xz} & 0 \\ 0 & 1 & 0 & 0 \\ -\varepsilon_{xz} & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$
$S_{67s} = \begin{bmatrix} 1 & 0 & 0 & x \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$	$S_{67se} = \begin{bmatrix} 1 & -\varepsilon_z(x) & \varepsilon_y(x) & \delta_x(x) \\ \varepsilon_z(x) & 1 & -\varepsilon_x(x) & \delta_y(x) \\ -\varepsilon_y(x) & \varepsilon_x(x) & 1 & \delta_z(x) \\ 0 & 0 & 0 & 1 \end{bmatrix}$

Transform D-H matrices into the Eqs. ((3)-(5)), then the precision machining constraint equation of CNC

camshaft grinder can be obtained,

$$\delta_x(z) + q_{5z}(\varepsilon_{xc} + \varepsilon_y(z) + \varepsilon_x(c) \sin c - \varepsilon_y(c) \cos c) - q_{5y}(\sin c + \varepsilon_z(x) \cos c + \cos c) - q_{5x}(\varepsilon_z(x) \sin c - \cos c - \varepsilon_z(c) \sin c) - x_w(\varepsilon_z(x) \sin c - \cos c + \varepsilon_z(c) \sin c) - y_w(\sin c + \varepsilon_z(x) \cos c + \varepsilon_z(c) \cos c) + z_w(\varepsilon_{xc} + \varepsilon_y(z) - \varepsilon_x(c) \sin c + \varepsilon_y(c) \cos c) \quad (6)$$

$$+ \delta_x(c) \cos c - \delta_y(c) \sin c = x + q_{8x} + x_t + \delta_x(x) + z\varepsilon_{xz} - q_{8y} \delta_z(x) - y_t \varepsilon_z(x) + q_{8z}(\varepsilon_{xz} + \varepsilon_y(x)) + z_t(\varepsilon_{xz} + \varepsilon_y(x))$$

$$\delta_y(z) - q_{5z}(\varepsilon_{yc} + \varepsilon_x(z) - \varepsilon_x(c) \cos c - \varepsilon_y(c) \sin c) - q_{5y}(\varepsilon_z(z) \sin c + \varepsilon_z(c) \sin c - \cos c) - q_{5x}(\varepsilon_z(x) \cos c + \sin c + \varepsilon_z(c) \cos c) + x_w(\varepsilon_z(z) \cos c + \sin c + \varepsilon_z(c) \cos c) - y_w(\varepsilon_z(z) \sin c - \cos c + \varepsilon_z(c) \sin c) - z_w(\varepsilon_{yc} + \varepsilon_x(z) - \varepsilon_x(c) \cos c + \varepsilon_y(c) \sin c) + \delta_x(c) \sin c + \delta_y(c) \cos c = y_t + q_{8y} + \delta_y(x) - (q_{8z} + z_t)\varepsilon_x(x) + (q_{8x} + x_t)\varepsilon_z(x) \quad (7)$$

$$z + \delta_z(z) + \delta_z(c) - q_{5z}\varepsilon_y(c) \cos c + q_{5y}((\varepsilon_{xc} + \varepsilon_y(z)) \sin c + (\varepsilon_{yc} + \varepsilon_x(x)) \cos c + \varepsilon_x(c)) + q_{5x}((\varepsilon_{yc} + \varepsilon_x(z)) \sin c - ((\varepsilon_{xc} + \varepsilon_y(z)) \cos c - \varepsilon_y(c)) + x_w((\varepsilon_{yc} + \varepsilon_x(z)) \sin c - (\varepsilon_{xc} + \varepsilon_y(z)) \cos c - \varepsilon_y(c))y_w((\varepsilon_{xc} + \varepsilon_y(z)) \sin c + (\varepsilon_{yc} + \varepsilon_x(z)) \cos c + \varepsilon_x(c)) + z_w = z + q_{8z} + z_t + \delta_z(x) - x\varepsilon_{xz} + (q_{8y} + y_t)\varepsilon_x(x) - q_{8x}(\varepsilon_{xz} + \varepsilon_y(x)) - x_t(\varepsilon_{xz} + \varepsilon_y(x)) \quad (8)$$

In the model of the linkage between the X axis and the C axis of machine tools, z' 、 z'' represent the camshaft outline curve, grinding wheel radius is R, the rotation angle of the grinding wheel is θ . According to the ref. [1], the position matrix of actual grinding point in workpiece branch and in tool branch can be obtained,

$$S_{8r} = \begin{bmatrix} -R \cos(\arcsin(\rho \sin(\theta - \varphi) / R)) & \rho \sin(\theta - \varphi) & z'' & 1 \end{bmatrix}^T$$

$S_{5r} = \begin{bmatrix} \rho \cos \varphi & -\rho \sin \varphi & z' & 1 \end{bmatrix}^T$, Where φ represent the cam rotation angle, z' 、 z'' represent the Z axis coordinate values of the grinding point P in the workpiece coordinate system and in the tool coordinate system, respectively.

The 21 error parameters in the precision machining constraint equation can be measured and identified by the dual ball bar instrument and multi-body system theory. By mapping the relationship between tool route and NC instruction, and between NC instruction and practical tool path, the NC instructions under the condition of error can be corrected. The machine tool after error compensation was chosen as the research object to process the automobile engine camshaft, the maximum error has been dropped from 22 μ m to 9 μ m, thus, the machining accuracy of the machine tool after error compensation have been significantly improved.

6 Conclusion

Based on the motion condition of the tilting head of five-axis machine tools, three measurement patterns in Y direction, X direction and Z direction are proposed respectively using DBB. Then, the relative displacement equations between two balls of DBB in measurement patterns in Y direction, X direction and Z direction are established respectively on the basis of HTM and MBS theory. Finally, the geometric error parameters of the tilting head (B-axis) are identified totally. The presented method in this paper is universal and provides a reference for the error identification for the tilting head of five-axis machine tools.

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