

# A Comparative Study of an Lpg- Spark Ignition Engine using Liquid Sequential Injection Technique

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**Abstract.** The development of liquid phase Liquefied Petroleum Gas (LPG) as an alternative fuel for the gasoline fuel is becoming popular in spark ignition engines. In this research, an experimental work was conducted on a 1.6L 4-cylinder multi-port injection engine. The engine was retrofitted with a Liquid Sequential Injection (LSI) LPG system. Its performance in terms of brake power, brake torque, brake specific fuel consumption (BSFC) and exhaust gas emissions (CO, CO<sub>2</sub>, HC, O<sub>2</sub>, and NO<sub>x</sub>) were evaluated from 1500rpm to 4000rpm with half-open and wide-open throttle positions. In comparison with gasoline, the brake torque and brake power for LPG were higher, but the BSFC and exhaust emission of CO and CO<sub>2</sub> for LPG are lower.

## 1 Introduction

Liquefied petroleum gas (LPG) is a clean alternative fuel for vehicle compared to the conventional fuels such as gasoline and diesel. In general, conventional fuels has high carbon ratio, low octane number and may gives higher emission levels compared to LPG [1, 2]. In year 2000, more than 6 million vehicles worldwide used LPG [3]. It showed that the technologies in LPG vehicles are getting good acceptance from automotive industries and the consumers. LPG has two phases; gas and liquid. From previous studies, LPG development is divided into five generations, which the 1st-4th generation used gas phase for the fuel delivery system while the latest system uses liquid phase.

According to the Material Safety Data Sheet (MSDS) of LPG in Malaysia, it contains 60% butane and 40% propane. Generally, the composition of LPG varies followed by season, country and characteristics of the supply crude oil, the refining process and the cost refined product. Therefore there is no constant standard value for the LPG composition [4]. Table 1 shows the composition of LPG as an alternative fuel of few countries. LPG is suitable as an alternative fuel for internal combustion engine (ICE). This is due to the higher Research Octane Number (RON) of LPG as compared with gasoline as

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shown in Table 2. The liquid LPG produced lower carbon emission due to the lower carbon content in LPG as compared to the gasoline fuel [5, 6]. The liquid LPG injection promised to give high resistance knocking phenomena and then the system can use a higher engine compression ratio thus the efficiencies of engine was increased [3].

The liquid phase is more effective than gas phase because a large amount of heat was absorbed into the intake air manifold due energy required to change the phase. This liquid phase is vaporized quickly, but at the same time it works as the cooler on of the mixture inside the intake manifold. As consequences, the engine received a greater mass of fuel-air mixture because greater density of cooler air and resulting volumetric efficiency and torque output were increased [4]. The purpose of this study is to evaluate the performance and emission of gasoline and liquid LPG as fuel of a 4-cylinder 1.6L displacement engine using multi-point port injection (MPI).

**Table 1.** Composition of LPG by country [7].

Country	Propane-butane (%) by volume
Belgium	50-50
France	35-65
Italy	25-75
Spain	30-70
United Kingdom	100-0
Germany	90-10
Malaysia	40-60

**Table 2.** Selected specifications of LPG and gasoline [8].

Property	LPG	Gasoline
RON	106-111	92-95
Specific Gravity	0.51	0.74
Stoichiometric A/F Ratio	15.7:1	14.7:1
Relative CO <sub>2</sub> per kJ	0.885	1.0

## 2 Experimental setup

All experiments were conducted on a *Proton Gen 2 (S4PH)* 1.6L vehicle. Table 3 shows the specification of the test vehicle engine. The fuel delivery type of this engine is a multi-point port injection (MPI) and a prototype liquid LPG injection were used. The LPG injectors were mounted on the intake manifold that close to the inlet valve. The fuel

pressure was regulated in the range of 9 to 10 bar to ensure that the LPG is in the liquid phase during injection process. The system was calibrated before running the experiments to ensure the quantity of injection LPG are adequate. The injection timing for LPG was emulated from the stock electronic control unit (ECU) and this system was controlled by the additional LPG ECU system.

The emissions of carbon dioxide ( $\text{CO}_2$ ), carbon monoxide (CO), hydrocarbon (HC), oxygen ( $\text{O}_2$ ) and oxides of nitrogen ( $\text{NO}_x$ ) emission were measured using *Autocheck* gas analyzer. At the same time the data of the air fuel ratio were also recorded. Next, the test vehicle was coupled to a *Dynapack* chassis dynamometer in order to measure the engine performance. The *Bosch* scan tool V1.2 management system was connected to the test vehicle during testing to enable real time data to be recorded and monitored for each test condition. The fuel consumption was measured using *Ono Sokki: FZ-2100* mass flow meter for both fuels used. A Schematic diagram of the experiment is shown in Fig. 1. The experiments were carried out at two different throttle valve positions; with half open throttle (HOT) valve position and wide open throttle (WOT) valve position. The engine speed for both throttle valve positions were tested from 1500 rpm until 4000 rpm with 500 rpm increment. The LPG used composes of 40% propane and 60% butane which were produced by Petroliam Nasional Berhad (PETRONAS).

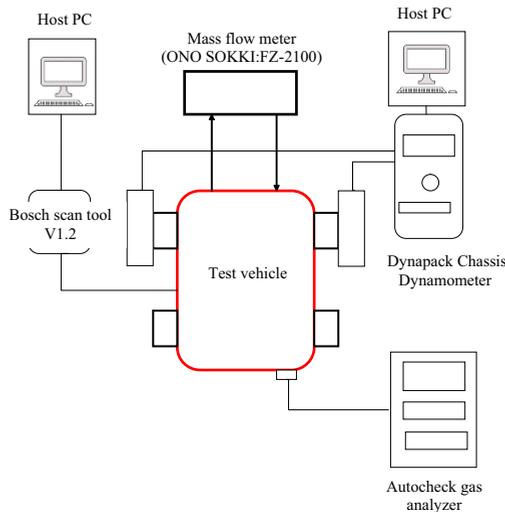
### 3 Result and Discussion

Fig. 2 shows the performance of both tested fuels at two different throttle valve positions, HOT and WOT valve position. The brake torque, brake power and brake specific fuel consumption were plotted against the engine speed. The brake power of LPG and gasoline were found increase for both HOT valve position and WOT valve position. LPG produced higher brake power at WOT valve position as compared to the gasoline, meanwhile at HOT valve position brake power produced by both of fuels showed nearly similar figure except at 4000 rpm where the brake power value of gasoline dropped. This is due to the fact that the LPG in liquid phase has a higher thermal efficiency than gasoline. Furthermore the temperature mixture was decreased and therefore the density and volumetric efficiency was inversely proportional condition to give positive effect [9].

The brake torque produced by LPG was compared with the gasoline fuel for both throttle valve position. The value of brake torque LPG was increase compared to the gasoline fuel. This is because the LPG cooled surrounding intake air inside intake manifold and results on the greater density [10]. BSFC variation of the tested fuels is shown in Fig. 2. For both throttle valve position, BSFC of gasoline was decreased with the increased of engine speed. Meanwhile, for the LPG, the BSFC was almost consistent throughout the engine speed. The BSFC of LPG was lower than gasoline except at 3500 rpm and 4000 rpm engine speed for tested throttle valve position. This is because the liquid LPG has higher pressure in fuel lines, hence the higher pressure fuel of LPG was released in the cylinder and resulted on lower fuel consumption of LPG itself as compared to gasoline fuel [8, 12].

**Table 3.** Specification of the test vehicle.

<b>Type model</b>	Proton Gen2 (S4PH)
<b>Total displacement</b>	1.6L
<b>Number of cylinders</b>	4
<b>Orientation</b>	East - West
<b>Valve train</b>	DOHC 16 V
<b>Compression ratio</b>	10.0: 1
<b>Bore x stroke</b>	76mmx 88mm
<b>Power</b>	82KW@6000 RPM
<b>Torque</b>	109lb-ft@ 4000 RPM
<b>Fuel/ system</b>	Gasoline/ multipoint injection

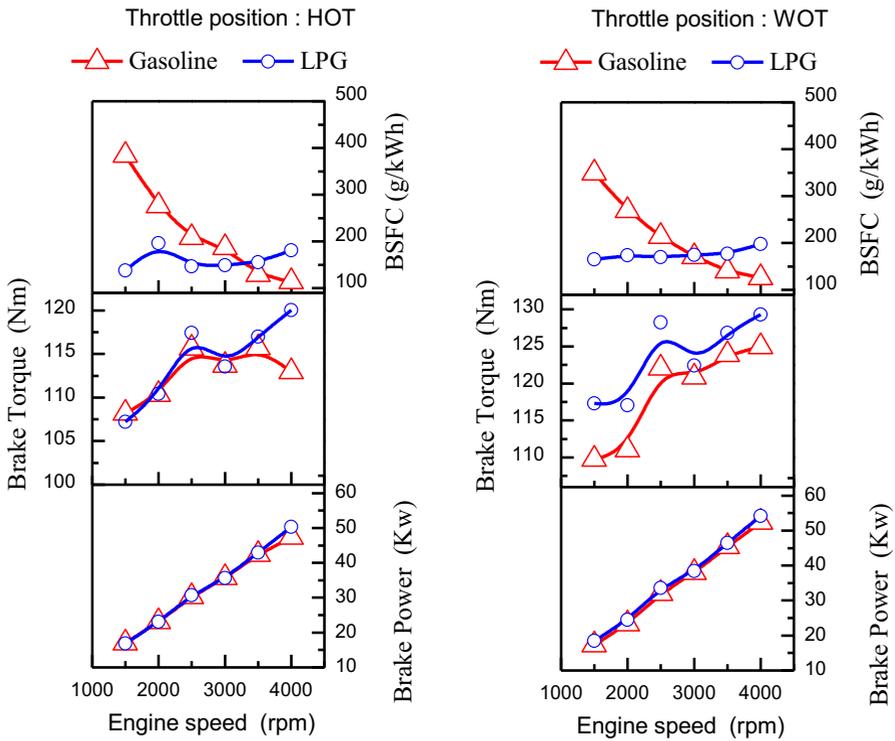


**Fig.1.** Schematic diagram of the experimental setup.

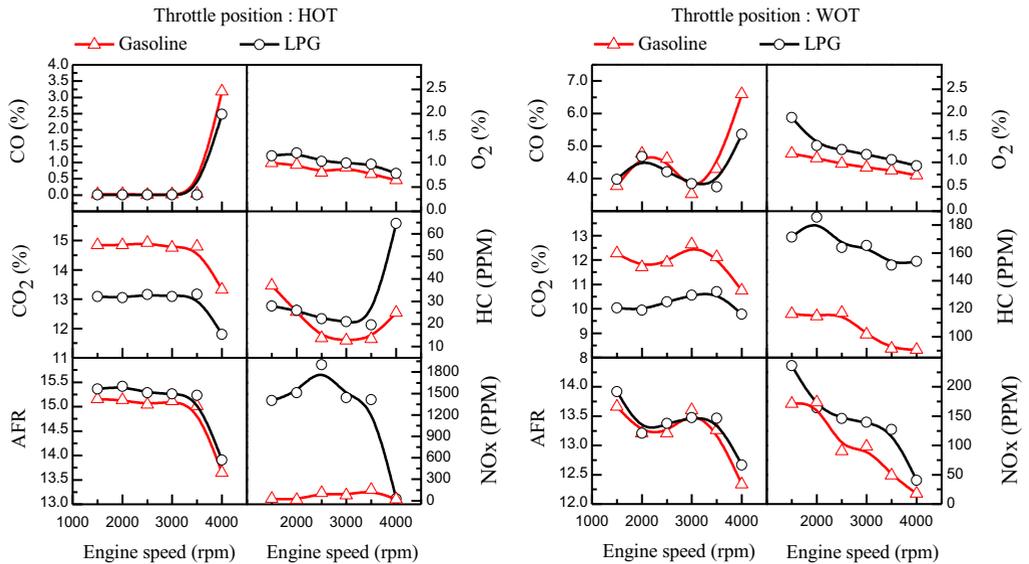
Fig. 3 shows the effects of emissions on SI engine fuelled by LPG and gasoline at HOT valve position to WOT valve position. The figure illustrated the emission of CO, CO<sub>2</sub>, HC, O<sub>2</sub>, and the NO<sub>x</sub>. From Fig 3, the CO emission of LPG at the both throttle valve position showed a similar pattern. At HOT valve position, the CO emission produced near similar value with WOT valve position, in the range of 1500rpm up to 3500rpm. However, at higher engine speed, 4000rpm, CO emission of gasoline yielded higher value compared to LPG. Since the carbon-hydrocarbon ratio of LPG is low and the LPG burn effectively with more homogeneous mixture[12]. The trend showed CO<sub>2</sub> emissions of gasoline produced higher value as compared to the LPG for the all engine speed tested. This was attributed to the fact that carbon-hydrogen ratio gasoline is largest value than the LPG [12]. In addition, gasoline has complete combustion compared with the LPG for these experiments. The HC emission of LPG was found higher than gasoline for both HOT valve position and WOT

valve position. This is due to gasoline burned completely compared with LPG. The HC emission produced at WOT valve position almost constant for both LPG and gasoline. The LPG was produced high HC compared with gasoline, this may be due to high temperature combustion and non-homogeneity of mixture [12].

The NO<sub>x</sub> emission of LPG at HOT valve position shows a different pattern as compared to the gasoline fuel and the maximum NO<sub>x</sub> produced was at 2500rpm as per shown in Fig. 3. At WOT valve position, the NO<sub>x</sub> production of LPG was higher than gasoline fuel but at 4000rpm the value is nearly zero. It showed that the LPG in liquid phase has higher peak temperature combustion that contributed the increased in NO<sub>x</sub> emission [1, 3, 13]. Throughout the experiments at WOT valve position, the AFR was maintained at lean condition until 3500rpm. Meanwhile at 4000rpm the AFR changed to rich condition for both of fuel. This is due to the ECU setting of vehicle used and SI engine technology itself. At WOT valve position, the AFR was remained at rich condition and the AFR value became richer as the engine speed increased.



**Fig. 2.** Performance of LPG and gasoline at different throttle position.



**Fig. 3.** Emissions of LPG and gasoline at different throttle valve position.

## 4 Conclusions

In a nutshell, performance of the liquid LPG injection was better than gasoline fuel. The BSFC for liquid LPG injection was significantly lower than gasoline fuel. The emission of CO was also lower as compared to the gasoline fuel. This result shows that LPG is capable to reduce the pollution of the environment. NO<sub>x</sub> emission produced by LPG is relatively higher than gasoline for both throttle valve position of which may due to the high temperature combustion in the engine cylinders. To reduce the NO<sub>x</sub> emission of liquid LPG injection, adjustment of the injection duration may be required in order to gives optimum amount of LPG injected in cylinder during combustion.

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